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COMPACT DRIVE, SPIROPLAN GEAR UNIT, AND METHOD FOR
MANUFACTURING A DRIVE UNIT

Description:

FIELD OF THE INVENTION

The present invention relates to a compact drive, a spioplan gear unit, and a method for manufacturing a drive unit.

5

BACKGROUND INFORMATION

[[DE]] German Published Patent Application No. 197 14 784

10 [[A1]] describes a compact drive, which includes an electric motor, at whose one end face a gear unit is situated arranged, and at whose other end face a frequency converter is situated arranged. In this case, the The electronics region and the motor region must be sealed with respect to the gear unit. In this context, it is disadvantageous that the axial length is large and that a power take-off can only be provided at one
15 end face of the compact drive.

JP-2002336305-A Japanese Published Patent Application No.

20 2002-336305 describes a gear motor, which does not include a frequency converter, however. Between the gear unit, motor, and brake, as well as between the corresponding housing parts, there are also interfaces that must be precisely manufactured and are therefore complex and expensive.

25 A frequency converter, which is axially mounted to an electric motor behind its blower and is therefore cooled by its air stream, is known from DE described in German Published Patent Application No. 196 22 396 [[A1]]. However, this therefore requires a large amount of space and, in addition, a fan that must be able to direct ambient air past.

30

A piezoelectric brake is ~~known from EP~~ described in European Published Patent Application No. 0 694 203 [[B1]].

[[DE]] German Published Patent Application No. 102 07 760

5 describes an adjusting gear, which, however, requires a large amount of space and an interface to the rotor of the motor.

[[DE]] German Published Patent Application No. 198 48 324

10 [[Al]] describes a coupling, which, however, requires additional, complicated and expensive machining of the rotationally mounted parts. In particular, not only are the sun and the rotor shaft included, but also a coupling sleeve. A higher number of parts also means higher storage costs.

15 [[EP]] European Published Patent Application No. 1 081 827

[[Al]] ~~relates to~~ describes an electric tool, in which a gear unit can be driven by an electric motor. However, an interface, which is difficult to manufacture, is provided between the housing parts of the motor and the gear unit.

20 An axially offset right-angle drive, i.e., a spiroid gear unit, which is connectible to an electric motor and can be driven by it, is ~~known from DE~~ described in German Published Patent Application No. 43 09 559.

25 SUMMARY

Therefore, ~~the object~~ Example embodiments of the present invention ~~is to further develop~~ may provide a compact drive ~~while eliminating~~ which may eliminate the above-mentioned
30 disadvantages. In particular For example, axial length ~~should~~ may be reduced and as many power take-off variants as possible ~~should~~ may be implementable, i.e. e.g., one-sided and two-sided power take-off.

~~This object of the present invention is achieved by the compact drive according to the features specified in Claims 1, 2, 3, or 4, by the spiroid gear unit according to the features specified in Claim 38, and by the method according to the features specified in Claim 39.~~

~~In the case of the~~ **For a** compact drive, ~~the essential features of the present invention are~~ **may include** that the compact drive includes at least three drive components, such as an electric motor, a gear unit, and an electronic circuit, in particular **e.g.**, a frequency converter; a central housing part being provided, and each drive component being surrounded by the central housing part and at least one housing cover of the respective drive component to form a respective housing.

~~In this context, it is advantageous that each~~ **Each** drive component ~~[[is]]~~ **may be** at least connected to the same central housing part for purposes of heat dissipation, and that the heat may therefore be distributed by it. ~~In particular~~ **For example**, the heat from a drive component, which is exhibiting a temperature peak at the time in question, may be introduced. Therefore, not only is each drive component's own housing utilizable for dissipating heat, but also the material region forming the housing for another component that is cooler at the same time. In the case of low motor currents and high speeds, heat flows, for example, from the drive unit to the motor; in the case of high motor currents and low speeds, the heat flows in the reverse direction. ~~In this context, the~~ **The** heat ~~must~~ **should** always travel through the central housing part and is distributed by it.

~~In the case of~~ **For** the compact drive, ~~the essential features of the present invention are~~ **include** that the compact drive includes at least three drive components, such as an electric motor, a gear unit, and an electronic circuit, in particular

e.g., a frequency converter; a central housing part being provided, the stator of the electric motor being detachably connected to the central housing part via, ~~in particular e.g.~~, a clamping joint. ~~In this context, it is advantageous~~ It may
5 be provided that the central housing part is initially machinable, using cutting and other machining steps; subsequently, the stator of the motor may then be inserted and locked. Therefore, no further machining step ~~[[is]]~~ may subsequently be necessary. ~~In this context, it is~~
10 ~~particularly advantageous that the~~ The motor ~~[[is]]~~ may be detachable again and exchangeable for purposes of maintenance or service. In addition, a slip joint is not necessary, since simple detachment is not implementable ~~in this case~~.

15 ~~In the case of~~ For the compact drive, the ~~essential features of the present invention are~~ include that it includes at least an electric motor, a brake, a gear unit, and a frequency converter, the output shaft of the gear unit and the rotor shaft being positioned in parallel to each other, the shaft-
20 center distance from at least one gear stage being determined, the first gear stage including a first toothed member connected to the rotor shaft, and a second toothed member, which engages with the first toothed member and is connected to an intermediate shaft; the brake, including at least one
25 brake-rotor shaft, being integrated in the housing of the compact drive, the brake-rotor shaft being parallel to the rotor shaft, and the brake-rotor shaft being connected to a toothed member, which engages with the second toothed member.

30 ~~In this context, it is advantageous~~ It may be provided that the overall axial length is reducible and one-sided and two-sided power take-off may be implemented. In addition, no axial length is necessary for the brake function, but rather the brake may be positioned in parallel next to the motor.

The action of the brake via the toothed members further allows the nominal braking torque to be increased or decreased.

~~In one advantageous refinement, the~~ The electric motor ~~[[is]]~~
5 may be a synchronous motor. ~~In this case, it is advantageous~~
It may be provided that high-speed positioning tasks may be
executed by the compact drive and/or that a high torque is
available over the entire speed range.

10 ~~In one advantageous refinement, the~~ The frequency converter
~~[[is]]~~ may be positioned laterally with respect to the rotor
shaft. ~~In this context, it is advantageous~~ It may be provided
that the overall length is reducible and the two sides of the
output shaft are accessible, i.e. e.g., two-sided power take-
15 off may be provided.

~~In one advantageous refinement, the~~ The gear region ~~[[is]]~~ may
be sealed with respect to the surroundings, and with respect
to the motor region and the electronics compartment. ~~In this~~
20 ~~context, it is advantageous~~ It may be provided that the gear
region may contain include lubricating oil, and that the
electronics and the stator and rotor parts remain protected
from the lubricant.

25 ~~In one advantageous refinement, the~~ The gear region, the
region of the motor, and the electronics compartment are may
be at approximately the same temperature. ~~In this context, it~~
~~is advantageous~~ It may be provided that no thermal barriers
are necessary, and that therefore, material may be dispensed
30 with and mass and costs may be reduced.

~~In one advantageous refinement, the~~ The motor includes may
include a sensor situated arranged at the one end of the rotor
shaft. ~~In this context, it is advantageous~~ It may be provided
35 that the compact drive may be used for positioning tasks, and

that the sensor is protected by the housing of the compact drive. A brake, which may also be protected by the housing of the compact drive, is connectible to the other end of the rotor shaft.

5

~~In a further advantageous refinement, the~~ The motor does may not include a sensor, but the position ~~[[is]]~~ may be ascertained with the aid of an estimation method. This allows may allow axial space to be saved.

10

~~Another considerable advantage of the present invention is that the~~ The rotor shaft ~~remains~~ may remain completely in the interior of the housing, and ~~that~~ therefore, no seals are may be necessary from the rotor shaft to the surroundings.

15 Consequently, a single shaft sealing ring ~~running~~ arranged on the rotor shaft is sufficient. Since the rotor shaft may have a high speed, the amount of heat generated is therefore much less than in the case of a motor having two shaft sealing rings, ~~in particular~~ e.g., on its two axial ends of the rotor
20 shaft.

The output shaft may have three shaft sealing rings. However, since the speed is much less than in the case of the rotor shaft, the entire amount of heat generated is less than in the
25 case of a design approach for the drive, where both the rotor shaft and the output shaft have two shaft sealing rings.

~~In one advantageous refinement~~ an example embodiment of the gear unit, at least one spur-gear stage is used, which means
30 that the overall axial length may decrease ~~decreases~~ and a solution optimal with regard to costs ~~[[is]]~~ may be produced.

~~In one advantageous refinement, the~~ The gear stage ~~[[is]]~~ designed may be arranged as a variable transmission having a
35 variable transmission ratio, which means that the wear of the

gear stage ~~[[is]]~~ may be minimized by the speed range, and the torque transmission ~~[[is]]~~ may be adjusted to the loading case. In the case of the variable transmission, it ~~is~~ advantageous may be provided that all of the seals for the engine compartment may even be dispensed with, since a variable transmission, ~~in particular~~ e.g., a continuously variable wide-belt transmission, requires no lubricant or only insignificant amounts of lubricant. Therefore, only seals from the interior of the compact drive to the external environment are may be necessary.

~~In one advantageous refinement, the~~ The rotor shaft and at least one shaft of the gear unit are may be supported in the same housing part. ~~In this context, it is advantageous~~ It may be provided that the shafts may already be accurately aligned with each other during the manufacturing and machining of the housing part, for the housing part may be finished during only one instance of chucking, and the relative position of the bearing seats may therefore be aligned in a very accurate manner.

The braking resistor and lubricant are connected in a manner allowing effective heat conduction, so that the lubricant may be heated by the braking resistor.

~~In one advantageous refinement, the~~ The heat transfer resistance from the braking resistor to the lubricant ~~[[is]]~~ may be less than that from the braking resistor to the environment. Therefore, when the ambient temperatures are low, the drive unit may be ~~advantageously~~ heated, in that the electronic circuit supplies sufficient power to the braking resistor. In this manner, the heat flows mainly from the braking resistor to the lubricant and therefore heats it sufficiently.

~~In one advantageous refinement, the~~ The heat transfer resistance from one of the stator windings to the gear lubricant agitated during operation ~~[[is]]~~ may be less than that from the stator winding to the environment. Therefore, the heat from the stator winding may be rapidly carried off through the central housing part to the lubricant. The latter is agitated and therefore transports the absorbed quantity of heat to its housing. Consequently, the heat is rapidly distributed. Only the operation of the drive unit ~~is~~ advantageously may be utilized for this purpose. Thus, a ~~further advantageous~~ an effect is that the central housing part, and also the housing cover of the gear unit, are brought to an approximately equal temperature, and therefore, the dissipation of heat to the environment ~~[[is]]~~ may be optimized since the heat is distributed over as much of the surface of the housing as possible to the environment. In addition, the central housing part and the lubricant provide a large, common heat capacity, which aids in absorbing the temperature peaks.

In the event of a low rotational speed and, ~~in particular~~ e.g., a large amount of power supplied to the motor, the resulting heat flows, on average, from the motor to the stator, and is then conducted through the housing to the environment.

In the event of high rotational speeds and, ~~in particular~~ e.g., a small amount of power supplied to the motor, the resulting heat flows, on average, from the stator to the motor, and is then conducted through the housing to the environment.

Since a heat barrier is ~~situated~~ arranged between the central housing part and the housing cover, which contains the electronic circuit, i.e., the frequency-converter electronics, the heat from the power semiconductors of the frequency-

converter output stage is carried off through this housing cover to the environment.

The braking resistor ~~is advantageously~~ may be connected in a housing pocket of the central housing part in a thermally conductive manner, and electrically connectible in a detachable manner, via a plug-and-socket connector, to the housing cover containing the electronic circuit.

10 ~~In one advantageous refinement, the~~ The compact drive is designed may be arranged to be a rectangular parallelepiped having 6 sides, recesses being able to be provided. ~~In particular~~ For example, only one to three sides are provided with housing covers for mounting and servicing the drive
15 components. Therefore, the other sides of the housing of the rectangular parallelepiped may be formed by the central housing part. Thus, the dissipation of heat to the environment ~~is then~~ may be improved as well, since the central housing part distributes the heat of the heat sources, such as
20 the braking resistor, brake, gear unit, and/or motor, etc.

To distribute the heat as mentioned above, it ~~is also~~ advantageous may be provided that each drive component is separated from every other one by only one housing wall of the
25 central housing part. Therefore, the heat ~~must~~ may need only to be passed through this one wall to another cooler component.

The electronic circuit is also connected to the one sensor,
30 which allows a temperature applicable to the central housing part to be determined. In this manner, the electronic circuit allows the temperature of the drive unit to be monitored and, ~~in particular~~ e.g., the flow of energy to be controlled. For the electronic circuit allows power to be supplied to the
35 braking resistor and heat to be supplied, and it allows the

motor to be operated with less power, i.e., the production of heat to be reduced. In the case of a providable brake coil of the brake, the supply of power for the purpose of dissipating heat is also controllable.

5

~~The essential features~~ **Features** of the spiroid gear stage are **include** that it is intended for a compact drive[[;]], a central housing part being provided[[;]], and each drive component being surrounded by the central housing part and at least one housing cover of the respective drive component to form a specific housing. ~~In this context, it is advantageous that because~~ **Because** of its cylindrical shape, the pinion of the spiroid gear stage ~~must~~ **may** no longer **need to** be adjusted and calibrated after assembly. In addition, the available axial offset in the spiroid gear unit allows the drive unit to be designed to be highly compact. ~~Surprisingly, a~~ **A** spiroid gear stage and a spur-gear power takeoff stage allow not only a higher total efficiency to be attained, but also a reduction in the length in the axial direction of the motor, since the spiroid gear wheel may be positioned in a direction perpendicular to it, and the power-takeoff gear wheel may therefore be provided closer to the pinion and the motor. A hypoid stage ~~would~~ **may** also provide similar ~~advantages~~ **aspects**, but the pinion would have to be adjusted after assembly. However, due to the use of the spiroid gear unit, the central housing part may be formed and guided around the gear unit [[in]] ~~such a manner,~~ that only a small gear housing cover [[is]] **may be** necessary, and that nevertheless, the assembly may be carried out simply and rapidly.

30

~~Further advantages are yielded from the dependent claims.~~

~~List of reference numerals~~

LIST OF REFERENCE NUMERALS

	1	bearing
	2	shaft sealing ring
5	3	housing cover
	4	cooling devices
	5	shaft sealing ring
	6	bearing
	7	shaft sealing ring
10	8	output shaft
	9	bearing
	10	gear wheel
	11	stator
	12	permanent magnets
15	13	rotor shaft
	14	pinion
	15	shaft sealing ring
	16	stator winding
	17	electronics compartment
20	18	bearing
	19	resolver stator
	20	bearing
	21	housing part
	22	housing part
25	23	resolver rotor
	31	electronics compartment
	40	gear unit
	51	electric motor
	52	first toothed member of the first gear stage
30	53	motor housing wall
	54	second toothed member of the first gear stage
	55	first connected toothed member of the second gear stage
	56	second toothed member of the second gear stage
	57	hollow output shaft
35	58	toothed member

59 brake
60 housing
61 housing wall
71 central housing part
5 72 housing cover
73 housing cover
74 housing cover
75 spiroid pinion
76 spiroid gear wheel
10 77 gear wheel
78 gear wheel
79 braking resistor

The Example embodiments of the present invention will ~~now be~~
are explained in greater detail below with reference to
figures: the appended Figures.

5 ~~An oblique view of a compact drive according to the present
invention is drawn in Figure 4, gear unit 40 only being
symbolically sketched.~~

BRIEF DESCRIPTION OF THE DRAWINGS

10 [[An]] Figure 1 is a perspective oblique view of a compact
drive according to an example embodiment of the present
invention ~~is drawn in Figure 1.~~

15 [[A]] Figure 2 is a partial cross-sectional ~~cut-off~~ view of a
compact drive according to an example embodiment of the
present invention ~~is shown in Figure 2.~~

20 ~~Shown in Figure 3 is a cut-off~~ partial cross-sectional view of
a compact drive according to an example embodiment of the
present invention, where, in contrast to Figure 2, the
frequency converter and the motor are situated arranged on
different sides of the output shaft.

25 Figure 4 illustrates an example embodiment of the present
invention.

Figure 5 illustrates an example embodiment of the present
invention.

30 Figure 6 illustrates an example embodiment of the present
invention.

DETAILED DESCRIPTION

35 In each instance, the gear unit ~~symbolically indicated~~
schematically illustrated in Figure 4 is implemented

differently in different example embodiment ~~variants~~ of the present invention. In a ~~first variant~~ an example embodiment, it is ~~designed~~ arranged to be a spur-gear unit, which is also ~~shown clearly~~ illustrated in Figures 2 and 3. In another
5 ~~variant~~ an example embodiment, the gear unit ~~from~~ illustrated in Figure 4 is ~~designed~~ arranged to be a variable transmission. This variable transmission may be manufactured in the form of a VARIMOT transmission of the company SEW Eurodrive, ~~i.e.~~ e.g., so as to have two disks rubbing
10 together, or in the form of a VARIBLOC transmission of the company SEW Eurodrive, ~~i.e.~~ e.g., as a continuously variable wide-belt transmission, the spacing of the two conical adjusting disks determining the transmission ratio. In a ~~further exemplary embodiment of the present invention~~, a A
15 chain may also be ~~advantageously~~ used in place of a v-belt.

In the exemplary embodiment ~~of the present invention according to~~ illustrated in Figure 2, the motor is positioned laterally with respect to the output shaft. Therefore, rotor shaft 13
20 and output shaft 8 are parallelly situated. The center-to-center distance of these shafts is determined by the engaging parts of the spur-gear stage, which ~~is made up of~~ includes a pinion 14 connected to rotor shaft 13 in a form-locked or friction-locked manner and a gear wheel 10, which is
25 manufactured as a spur gear and is connected to output shaft 8.

The compartment of the gear unit, ~~i.e.~~ the spur-gear stage, is sealed with respect to the compartment of the electric
30 motor. Shaft sealing ring 15 seals these compartments at the rotor shaft, since the rotor shaft carries permanent magnets 12 in the compartment of the motor, as well as pinion 14 in the compartment of the gear unit. Shaft sealing ring 5 seals the compartment of the gear unit with respect to the

compartment of the motor at output shaft 8, which is manufactured as a hollow shaft.

~~In a further exemplary embodiment of the present invention, a~~
5 A different gear unit ~~containing~~ including several gear stages may be used instead of the spur-gear stage ~~shown~~ illustrated.

~~In a further exemplary embodiment of the present invention,~~
10 ~~the~~ The output shaft ~~does~~ may not take the form of a hollow shaft, but rather a solid shaft. In addition, it is also possible to ~~design~~ arranged the output shaft according to the standard for robot interfaces, which means that a highly compact power take-off having a short overall axial length is produced.

15 Output shaft 8 is supported by bearing 1 in the same housing part 21, in which rotor shaft 13 is also supported by bearing 18.

20 The compartment of the motor is sealed with respect to the environment, using the shaft sealing ring 2 that ~~runs~~ is on output shaft 8 and is inserted into housing cover 3.

Housing parts 21 and 22 are provided with cooling devices 4
25 for dissipating the heat generated in the motor, gear unit, and frequency converter.

Output shaft 8 is supported, ~~in turn,~~ by the other axially opposite bearing (6, 9) in the same housing part 22, in which
30 rotor shaft 13 is also supported by the other bearing 20.

~~A considerable advantage of the compact drive is that no~~ No
clutch is necessary between the motor and the gear unit, which consequently eliminates the need for additional parts. In
35 particular, the motor and gear unit even use the same housing

parts jointly. In addition, it is possible to already accurately align the shafts with respect to each other during the processing and machining of the housing part, in that the relative position of the bearing seats for the motor and the gear unit, e.g., ~~in particular~~ of bearings 9 and 20, may be set in an extremely accurate manner during manufacturing, for the housing part may be finished in only one machine tool, in only one instance of chucking. Therefore, the relative position of the bearing seats may be adjusted very accurately with respect to each other. The common usage of a housing part ~~[[is]]~~ may also be advantageous in that, in this manner, the compact drive not only requires a small volume, but also has a particularly high strength, since the forces of the motor and the gear unit are transmitted to each other inside the same housing part.

A further ~~advantage~~ aspect of a jointly used, i.e., central housing part is also the rapid distribution of the heat supplied in one region to other regions. Therefore, no interfaces hinder the transport of heat in the direction of other regions. Since the different regions always produce different portions of the heat, the heat produced by the largest, corresponding source may be distributed more rapidly to the other regions via the central housing part. This distribution of the heat ~~has the further advantage~~ provides that a very large surface area is available for dissipating heat to the environment. Therefore, even special, complicated, and expensive cooling devices, such as cooling fingers and/or cooling fins, may be eliminated. In addition, the housing may be manufactured to be essentially smooth, which means that fluids may drain off rapidly and easily.

The compartment of the gear unit is sealed with respect to the environment, using the shaft sealing ring 7 that ~~runs~~ is on output shaft 8 and is inserted into housing cover 22.

Stator 11 having stator windings 16 is positioned around rotor shaft 13.

5 This electric motor is a multiphase synchronous motor. However, ~~in other exemplary embodiments of the present invention,~~ any other motor may be integrated into the compact drive in place of the synchronous motor.

10 Shaft sealing ring 15, which ~~runs~~ is on the rotor shaft and is inserted into housing part 22, seals the compartment of the gear unit with respect to the compartment of the motor.

Electronics compartment 17 for the frequency converter is not
15 sealed with respect to the compartment of the motor.

On its one axial end, the motor supports a resolver, which includes a resolver stator 19 and a resolver rotor 23.

20 ~~In other exemplary embodiments of the present invention, other~~ Other angular-position sensors or angular-velocity sensors may also be provided in place of the resolver. ~~In other exemplary embodiments of the present invention, a~~ A brake may also be integrated into the compact drive on the side opposite to the
25 angular-position sensor.

~~In other exemplary embodiments of the present invention, the~~ The frequency converter is operated, ~~in turn, in such a manner~~ that with the aid of a method, the angular value is estimated
30 using a suitable motor model. This allows the overall axial length to be further reduced.

~~Another variant of an~~ An exemplary embodiment according to the present invention is ~~shown~~ illustrated in Figure 2, where
35 electronics compartment 31 is not directly to the compartment

of the motor, but rather output shaft 8 ~~lies~~ is between them.
In this example embodiment, shaft sealing ring 5 ~~then~~ seals
the compartment of the gear unit with respect to electronics
compartment 31, shaft sealing ring 5 running on output shaft 8
5 and being seated in housing part 21.

The gear unit may be filled with lubricant, such as
lubricating oil, lubricating grease, ~~or the like~~ etc.

10 In the ~~shown~~ illustrated exemplary embodiments ~~according to~~
~~the present invention~~, no substantial thermal barrier is
provided between the compartments of the frequency converter,
i.e., the electronics compartment, and the gear-unit
compartment and the motor compartment. Consequently, the
15 compartments are at approximately the same temperature.
Approximately the same temperature means, e.g., a maximum
temperature difference of 10°C during continuous operation at
nominal load. ~~Of course, a~~ A larger, short-term temperature
difference of the compartments is attainable in the case of
20 intermittent operation. ~~What is advantageous and surprising~~
~~about this design is that no~~ No special thermal barrier is
necessary, and that the amount of material, mass, and costs
may therefore be reduced.

25 ~~In other exemplary embodiments of the present invention,~~
~~thermal~~ Thermal barriers may also be provided between two or
more of the compartments.

~~In other exemplary embodiments of the present invention, the~~
30 The motor ~~is designed~~ may be arranged to be multipolar, in
particular e.g., eight-poled or ten-poled. The motor ~~is~~
~~advantageously designed according to DE~~ may be arranged as
described, for example, in German Published Patent Application
No. 100 49 883 or [[DE]] German Published Patent Application
35 No. 103 17 749. Therefore, a single gear stage, together with

such a multiphase motor, is sufficient to cover a wide range of transmission ratios.

~~In other exemplary embodiments of the present invention, not~~

5 ~~Not~~ a hollow shaft, but rather a cylindrical shaft stub ~~takes~~
may take the form of an output shaft, this output shaft being connectible to the device to be driven, using a feather-key connection.

10 ~~In further exemplary embodiments of the present invention, the~~
The output shaft and the output-side housing part are may be
manufactured in accordance with robot interface EN-ISO 9402-1. This allows the overall axial length to be reduced and a high torque to be transmitted. In addition, compatibility with
15 corresponding devices to be driven and connected is achieved.

The electrical connection terminals are provided on the back of the housing ~~and are therefore not visible in Figures 1 through 4.~~ However, other positions for the connection
20 terminals may also be provided ~~in further exemplary embodiments of the present invention.~~

~~In further exemplary embodiments of the present invention, the~~
The connection terminals are ~~only designed~~ may be arranged as
25 a power supply. ~~In particular~~ For example, only electric power cables are run to the compact drive. In this context, the transmission of data to the frequency converter or from the frequency converter to another, ~~in particular~~ e.g., superordinate unit is accomplished by modulating them upon the
30 power lines, the transmission of data being necessary for the data communication. The modulation may be accomplished in a known conventional manner, ~~in particular~~ e.g., as known conventional from powerline communication or according to FSK or the FH/PSK method, i.e., Frequency Hopping Phase Shift
35 Keying.

~~Shown~~ Illustrated in Figure 5 is a ~~further~~ an exemplary embodiment of the present invention, in which housing 60 has a compartment for electric motor 51, the compartment being separated off by a housing wall 53. The electric motor has a pinion as a first toothed member 52 of the first gear stage, the pinion being connected to the rotor shaft of electric motor 51 in a form-locked or force-locked manner and engaging with a second toothed member 54 of the first gear stage, the second toothed member being connected on an intermediate shaft. This intermediate shaft is connected, ~~in turn,~~ to a further toothed member, ~~namely~~ e.g., first toothed member 55 of the second gear stage, the first toothed member engaging with second toothed member 56 of the second gear stage. This toothed member 56 is connected to hollow output shaft 57, which allows a compact connection to a device to be driven. ~~It is now important that a~~ A brake 59 may include a brake rotor, which is connected to a pinion in the form of a toothed member 58, the pinion engaging with the second toothed member of the first gear stage. Thus, electric motor 51 is manufactured without a brake. Brake 59 carries out its braking action on a toothed member, via pinion 58. The braking action is carried out, ~~as it were,~~ not directly, but rather indirectly.

~~In the specific embodiment according to~~ As illustrated in Figure 5, ~~it is also important that~~ the gear region ~~[[is]]~~ may be sealed with respect to the region of the electric motor and the brake. The regions of the brake and the motor do not have to be sealed with respect to each other. The regions are arranged in corresponding housing pockets, whose walls are the housing walls. The electronics region, i.e., the frequency-converter region, may also be sealed with respect to the other regions, ~~in particular~~ e.g., with respect to the gear-unit compartment. No absolute sealing is necessary from the

electronics region to the motor region. The cable bushings from the electronics region to the motor region, and in some instances to the gear region, are sealed and manufactured to provide a high degree of protection.

5

The gearing parameters in Figure 5 may be selected so that the braking torque of the brake transmitted to the rotor shaft is less than, greater than, or equal to the rated motor torque. ~~In particular~~ For example, in a ~~design~~ an arrangement in which
10 the transmitted braking torque is greater than the rated motor torque, the brake may be manufactured to be very small and compact. This allows the overall volume to be reduced.

In Figures 1 ~~through~~ to 5, it ~~is clearly~~ should be apparent
15 that the housing of the compact drive ~~essentially has~~ may have the shape of a right parallelepiped. Since the axis of the hollow output shaft is oriented in the direction of the normal to the largest surface of the rectangular parallelepiped, the drive unit may be used in applications that are usually
20 reserved for right-angle drives. The compactness of the drive unit ~~according to the present invention is especially~~ may be produced by the configuration of the rotor axis, namely oriented in a direction parallel to the output shaft of the compact drive.

25

~~In further exemplary embodiments of the present invention,~~
instead Instead of the block-shaped design arrangement, a
similar ~~design~~ arrangement of the housing ~~is also advantageous~~
may be provided, although the rectangular parallelepiped has a
30 sizable recess in the region of the output shaft. This does reduce the spacing of the output bearings and, therefore, the rigidity with respect to transverse forces, but there is sufficient space for connection to the shaft to be driven.

In addition, the compact drive ~~according to the present invention allows~~ may allow a higher efficiency to be attained than in the case of right-angle motors, since spur-gear units may be used. This is because these always have a higher efficiency than right-angle drives.

~~In the present invention, it is~~ It may also very ~~important~~ be provided that the housing of the entire compact drive has a central housing part, which fills the housing function for the motor, gear unit, and electronics. Only housing covers are connected on the housing part. The central housing part includes bearings for the gear unit and for the motor. It is also in contact with gear lubricant on its inner side. In addition, the laminated core of the stator of the motor is supported in the central housing part. However, not only are the gear lubricant and the stator windings connected to the central housing part so as to effectively conduct heat, but also the electronics, including an optional braking resistor. However, in the part of the electronics that does not include the braking resistor, a housing cover is connected so as to conduct heat, in order to dissipate the heat of the electronics to the environment.

Since the central housing part is at least connected to all three drive components so as to effectively conduct heat, and therefore, each drive component may give off heat to the central part. Since the central part is very large, it has a high heat capacity and may therefore absorb short-term peaks of the heat flowing from the drive components to it. This ~~[[is]]~~ may be particularly advantageous, since the different drive components have different heat losses at different operating states.

For example, the heat generation of the motor is very high when the drive unit is started from rest in the range of the

breakaway torque. Stated in general terms, the heat generation of the motor is a function of the torque. However, the heat generation of the gear unit is a function of rotational speed, i.e., it is only relevant later, with increasing speed.

Therefore, an ~~advantage~~ **aspect** of the common, central housing part is that a large mass is available and, therefore, effective heat equalization takes place between the drive components.

Because of the central housing part, interfaces between the drive components are ~~then~~ eliminated[[;]], i.e., fewer components, such as housing parts, seals, screws, and the like **etc.**, are necessary, and the entire drive unit may be manufactured more compactly. ~~In this context, it is also important that the processing~~ **Processing** steps usually necessary for interfaces are eliminated.

~~In further exemplary embodiments according to the present invention, toothed~~ **Toothed** members of gears of a gear-unit construction kit are **may be** reused for gear stages of the compact drive. Therefore, the total number of toothed members to be manufactured is less.

~~In further exemplary embodiments of the present invention, a~~ **A** sensor [[is]] **may be** mechanically connected to the output shaft and electrically connected to the electronics region. The sensor is an angular-position sensor and/or a torque sensor. Therefore, the control software in the frequency converter may be manufactured to have especially good control characteristics.

~~In a further advantageous refinement, the~~ **The** compact drive **has may have**, on the front side and on the back side, the same

mechanical interface for connection to a device to be driven.
To this end, the output shaft is manufactured as a hollow
shaft that passes through from the front side to the back
side. Therefore, the compact drive may be installed in a
5 system or machine from both sides. In addition, it is also
possible to connect compact drives in series, which means that
the total torque may be increased.

~~In a further advantageous refinement, the~~ The electric motor
10 ~~[[is]]~~ may be a synchronous motor. Therefore, the losses due
to slip are reducible and the control accuracy is improvable.

~~In another advantageous refinement, the~~ The electric motor may
be manufactured as a reluctance motor in a cost-effective
15 manner.

~~In other exemplary embodiments of the present invention,~~
~~instead~~ Instead of the brake, a different energy-storage
mechanism ~~[[is]]~~ may be manufactured, i.e., a flywheel or
20 another rotating mass, for example. To this end, the gear
teeth of the brake pinion are designed together with the gear
teeth engaging with them, ~~in such a manner,~~ that the flywheel
rotates at a high speed, ~~in particular~~ e.g., for saving space.

~~In other exemplary embodiments of the present invention, the~~
25 The pinion of the brake engages may engage with the first
toothed member instead of the second toothed member and
therefore performs the braking action directly.

~~In other exemplary embodiments of the present invention, a~~ A
30 gear stage, ~~in particular~~ e.g., the one furthest to the side
of the power take-off, ~~takes~~ may take the form of a right-
angle gear stage, ~~in particular~~ e.g., a worm-gear stage or a
spiroid gear stage. This ~~renders~~ may render particularly
35 compact designs arrangements possible.

~~In an advantageous refinement of the present invention, the~~
The electronics region borders may border on the gear region
and ~~[[is]]~~ may only be separated from it by a housing wall.

5 Therefore, the heat generated by the electronics may be
dissipated via the housing wall and the lubricant of the gear
unit. The lubricant circulation, which is increased by the
movement of the toothed members, is therefore an ~~important~~
~~means~~ arrangement for dissipating the heat to the environment,
10 for the heat is transported by the lubricant to the other
housing walls of the gear region, and given off to the
environment from there. Heat may be removed in the same
manner, when the motor region borders on the gear region via a
housing wall.

15 As ~~can be clearly seen~~ illustrated in Figures 2 ~~through~~ to 5,
the region of the electronics is near the region of the motor.
Therefore, a transmitter or a plurality of sensors may be
provided in the electronics compartment, ~~in particular~~ e.g.,
20 on a printed circuit board of the electronics themselves.
Depending on the manner of attachment of the sensor to the
printed circuit board, this sensor therefore detects values of
state variables in the electronics or motor compartment. For
this purpose, at least a subsection of the transition of the
25 electronics region to the motor region is ~~designed~~ arranged to
be open. Therefore, if the housing cover is mounted, along
with the electronics, on the central housing part, then the
sensor connected to the electronics is correctly positioned.

30 On one hand, temperature sensors and, on the other hand,
angular-position sensors or angular-speed sensors may be
~~advantageously~~ provided as sensors. Hall sensors, which
detect the magnetic field of the permanent magnets of the
rotor, may be used, for example, as angular-position sensors.

In this manner, the angular-position data and/or angular-speed data may be acquired directly at the rotor.

When a temperature sensor is used, it is possible to monitor the temperature. However, it is also possible to control the temperature by specifying the power supplied to the braking resistor and/or the power transmitted to the motor.

Therefore, it is ~~of course possible, all in all,~~ to even control the temperature.

To obtain a high torque, the stator windings are manufactured as tooth windings, i.e., windings are slid onto each tooth or wrapped around it to obtain, ~~in particular~~ e.g., higher torques and higher efficiencies.

Additionally provided in the region of the electronics is a housing wall of the central housing part, the braking resistor being attachable to the housing wall ~~[[in]] such a manner,~~ that it is connected to the housing wall and the gear lubricant located on the other side of this wall so as to be able to conduct heat highly effectively. ~~In further exemplary embodiments of the present invention, a~~ A housing pocket ~~[[is]]~~ may be formed for accommodating the braking resistor. The braking resistor allows the compact drive to be heated in a manner that surpasses the options for heating the stator windings with the aid of a superimposed d.c. component. Therefore, the compact drive may also be used at very low temperatures.

In addition, an electronic type label, i.e., a writable memory that stores data about the drive, as well as status data or maintenance and diagnostic information, is provided in the region of the electronics. The storing may also be carried out without an external power supply. ~~In particular~~ For example, data that relate to temperature, rotational speed,

and/or torque may be stored. In this context, the storing and/or summing of the time periods weighted with the respective temperature may be used for determining the lubricant-replacement interval.

5

~~In other exemplary embodiments of the present invention, the~~
The core assembly ~~includes~~ may include teeth, onto which the stator windings are slid, or around which the stator windings are wound. Therefore, high torques and a high motor efficiency ~~are advantageously~~ may be attainable with low manufacturing costs.

10

Figure 6 ~~shows a further~~ illustrates an exemplary embodiment of the present invention, in which no brake is provided and the output shaft of the drive unit is perpendicular to the rotor shaft. This produces a compact right-angle drive. Central housing part 71 fulfills the function of forming the housing for the electronics region, the motor region, and the gear region.

15

20

Housing cover 72 includes a fixture for a printed circuit board, which is connectible to a further printed circuit board with the aid of a plug-and-socket connector, the further printed circuit board being accommodated and connected in central housing part 71.

25

Housing cover 73 is removable, which means that the motor, including the stator core assembly, stator windings, and also the rotor shaft, may be installed in central housing part 71.

30

Housing cover 74 is removable as well, which allows the gear unit to be installed and/or lubricant to be poured in.

The gear unit has two stages, the input stage being a spiroid-gear stage, and the output stage being a spur-gear stage.

35

Spiroid pinion 75 connected to the rotor shaft of the motor has a cylindrical contour and is axially offset from the axis of the spiroid gear wheel 76 engaging with pinion 75. The output stage is formed by gear wheels 77 and 78.

5

~~An especially~~ A compact drive unit, ~~in particular e.g.,~~ a frequency-converter gear motor, may be implemented by the ~~specific~~ example embodiment shown.

10 The axial offset of the spiroid input stage allows a lesser straddling height to be maintained. This means, for example, that the output shaft may drive a wheel, which supports and drives a belt that runs under the compact drive ~~of the present invention~~ and is not disturbed by the motor housing, for the
15 axial offset allows the motor to be positioned higher than the bottom edge of the gear unit. In this context, the electronics region is "at the top" and the bottom edge of the gear unit is "at the bottom" in Figure 6.

20 If a bevel-gear unit ~~were to be~~ is used as an input stage, then only a motor having a lower output and, therefore, smaller construction type ~~would~~ may be usable. However, ~~the present invention allows~~ a high-torque motor ~~[[to]]~~ may be used, since the axial offset provides room without reducing
25 the straddling height.

The optimization of the straddling height allows the distance from wheels 78 and 76 to the bottom edge of the gear unit to be at as short as possible.

30

In the motor region, the central housing part is higher than the lower side of the gear unit, i.e., the bottom edge of the gear unit, the bottom edge of the gear unit being provided on the side facing away from the electronics region.

35

In the spiroid gear stage, the pinion axis does not intersect the wheel axis and is oriented perpendicularly to it, the axial offset being less than the pitch-circle radius of the gear teeth of the wheel. This allows high efficiencies to be attained, even though an axial offset is provided. The contour of the pinion is a cylinder, i.e., not a cone. This is in contrast to bevel-gear stages.

Braking resistor 79 is ~~situated~~ arranged in a recess of central housing part 71, the recess extending into the gear region. Therefore, the heat passing through the housing wall of the housing part may be distributed rapidly and easily by the lubricant, which means that the heat may be distributed rapidly via the central housing part. ~~In this context, it is also particularly advantageous that in~~ the vicinity of the input side, i.e., closer to the input side than the output side, the housing pocket extends into the gear region, for the movement of the toothed members, and therefore the speeds of motion of the lubricant as well, are more rapid here, which results in rapid dissipation of the heat of the braking resistor.

A position sensor may be integrated in housing cover 73. However, ~~variants of the present invention that are even more advantageous allow~~ a sensor ~~[[to]]~~ may be eliminated and replaced by a sensorless, open-loop and closed-loop control method in the frequency converter, the control method being suitably implemented for, ~~in particular~~ e.g., synchronous motors. In this manner, housing cover 73 may be manufactured to be smaller.

~~Abstract~~

ABSTRACT

[[A]] In compact drive, spiroid gear unit, and method for manufacturing a drive unit that includes at least an electric
5 motor, a brake, a gear unit, and a frequency converter, the output shaft of the gear unit and the rotor shaft ~~being~~ are positioned in parallel to each other, and the shaft-center distance ~~being~~ is determined by at least one gear stage, ~~the~~.
10 The first gear stage ~~including~~ includes a first toothed member connected to the rotor shaft, and a second toothed member, which engages with the first toothed member and is connected to an intermediate shaft, the brake, including at least a brake-rotor shaft, being integrated in the housing of the compact drive, the brake-rotor shaft being parallel to the
15 rotor shaft, and the brake-rotor shaft being connected to a toothed member, which engages with the second toothed member.